

Analysis of Coil Spring Used In Two Wheeler Suspension System

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Abstract— The present work is carried on modeling, analysis and testing of suspension coil spring is to replace the existed coil spring used in popular two wheeler vehicle. The deflection of the coil spring is going to be reduced by using the new material. Static analysis determines the stress and deflection of the coil spring in finite element analysis. The testing prototype is used to test the spring under different loading conditions. FEA are the methods of finding approximate solutions to a physical problem defined in a finite region in this finite element analysis values are compared to the experimental values. A typical two wheeler suspension spring is chosen for study, analysis is carried out on ANSYS 16.2

Keywords: spring constant, modeling, ANSYS 16.2

I. INTRODUCTION

When people think about automobile performance, they normally think of horse power, torque and 0-100 km ph acceleration. But all of the power generated by a piston engine is useless if the driver can't control the bike. That's why the suspension system in an automobile is important and so much attention is given to it. The vehicle suspension system is responsible for the vehicle control, driving comfort and safety as the suspension carries the vehicle body and transmits all the forces between the road and the body.

Spring: A spring is defined as an elastic body, whose function is to distort when loaded and when the load is removed it regains its original shape.

Types of spring:

1. Helical spring
2. Torsional spring
3. Laminated or leaf spring
4. Special purpose spring

II. PROBLEM STATEMENT

Objectives of project are,

- Modeling, analysis & testing of coil spring used in two wheeler.
- Design and analysis of stress & deflection of coil spring.
- To test the spring under different loading condition.
- Use FEA method.

III. METHODOLOGY

- In this work ,modeling of new spring and analysis of spring material in order to optimize load carrying capacity of selected spring.
- Material select for analysis are vanadium chrome steel, oil tempered low carbon steel and Inconel X 750 alloy steel.
- The modeling of spring is to verify by using ANSYS 16.2 software to perform finite element analysis.
- The calculated results are to compare on common scale and the prediction of the spring can be conclude.
- Specification of coil spring Are as follows:

Table No. 3.1 Specification

Sr no.	Specification	Value
1	Material name	EN 42 steel
2	Outside diameter	49mm
3	Inside diameter	35mm
4	Mean diameter	42mm
5	Wire diameter	7mm
6	Spring index	6
7	Free length	230mm
8	Solid length	122mm
9	No of active coils	17
10	No of total coils	19
11	Density	78000kg/m ³
12	Young's Modulus	210000Mpa
12	Modulus of rigidity	80000Mpa

3.1 Equations of spring design:

$$1. K = Gd^4 / 8D^3N$$

$$2. \delta = P/K$$

Where,

K= stiffness of spring (N/mm)
G= modulus of rigidity (N/mm²)
D= mean coil diameter (mm)
N= no of active coils
 δ = deflection of coil (mm)
p= Load on coil (N)

3.2 Material and properties of spring design:

Table No 3.2 Material properties

Material's	Modulus of Elasticity (Mpa)	Modulus of Rigidity (Mpa)	Poisson's Ratio (Mpa)

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EN42	210	80.7	0.3
Vanadium Chrome Steel	207	81.4	0.27
Oil Tempered Steel	207	80.2	0.29
Inconel X750	214	82.9	0.29

IV. ANALYSIS OF COIL SPRING BY THEROROTICALLY

Table No. 4.1 load vs deflection

Material	Deflection at various loads			
	AISI	ASTM	ASTM	AMS
load	1074	A231	A229	5698
30kg	15.304	15.172	15.399	14.898
60kg	30.608	30.345	30.799	29.797
90kg	45.912	45.518	46.199	44.696
120kg	61.210	60.691	61.590	59.594
150kg	76.521	75.864	76.998	74.493

V. FEA ANALYSIS OF COIL SPRING

5.1 Introduction to FEA:

A primary reason to use FEA in coil design is to reduce the error caused by simplifications of the equations. The most accurate FEA results can be obtained by creating 3-D parts of a coil spring, followed by using finer meshing of the parts with 3-D solid element.

Steps used in analysis of coil spring by FEA:

1. Preprocessing.

Preprocessing consist of preparation of data such as nodal coordinate, connectivity, boundry conditions, loading and material information.

2. Processing.

It involves stiffness generation, stiffness modification, and solution of equation, resulting in the evaluation of nodal variable.

3. Post processing.

It deals with the preparation of result.

5.2 Material:

1. EN 42 (AISI 1074)

Fig 1, 2,3,4,5 shows the load vs deflection diagram for various load 150 kg, 120 kg, 90kg, 60 kg, 30kg respectively.

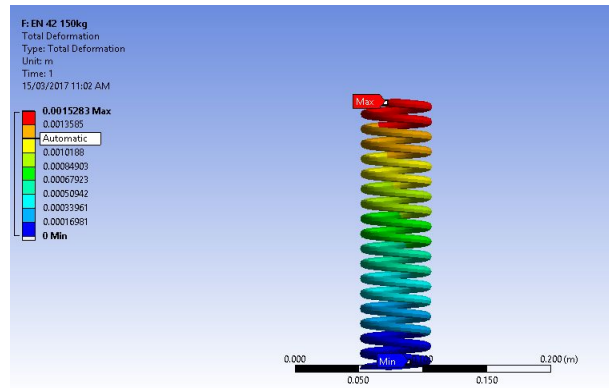


Fig.1 load (150 kg) vs deflection.

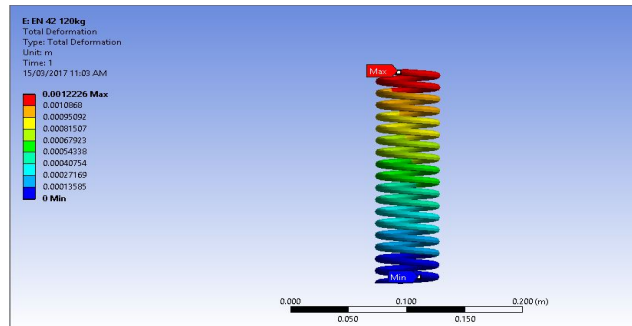


Fig.2 load (120 kg) vs deflection.

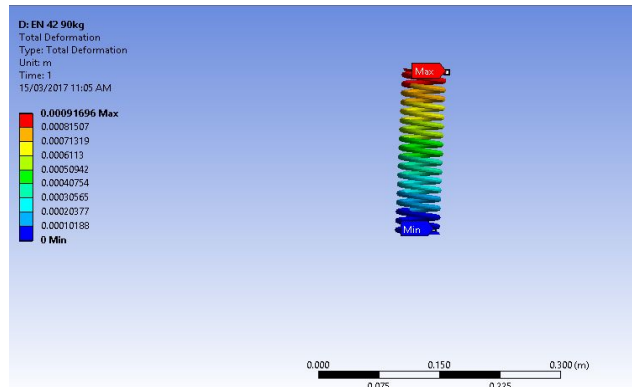


Fig.3 load (90 kg) vs deflection.

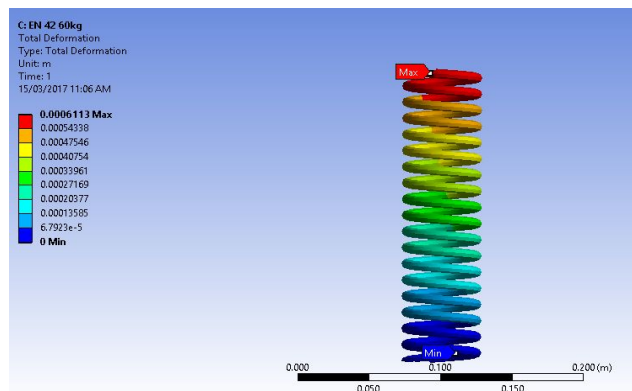


Fig.4 load (60 kg) vs deflection.

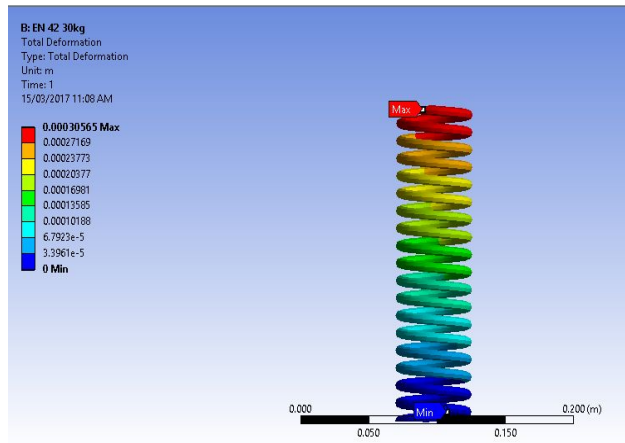


Fig.5 load (30 kg) vs deflection.

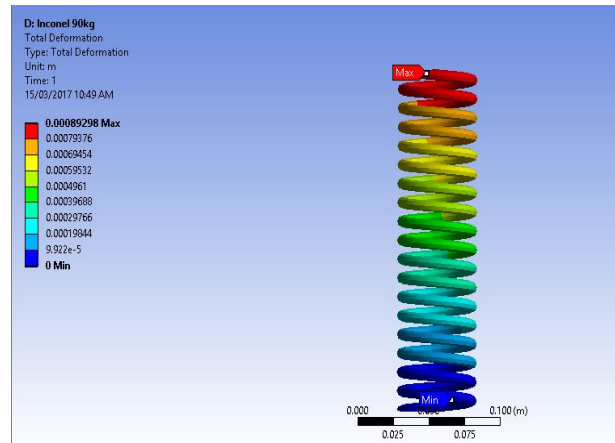


Fig.8 load (90 kg) vs deflection.

2. Inconel X 750 (AMS 5698)

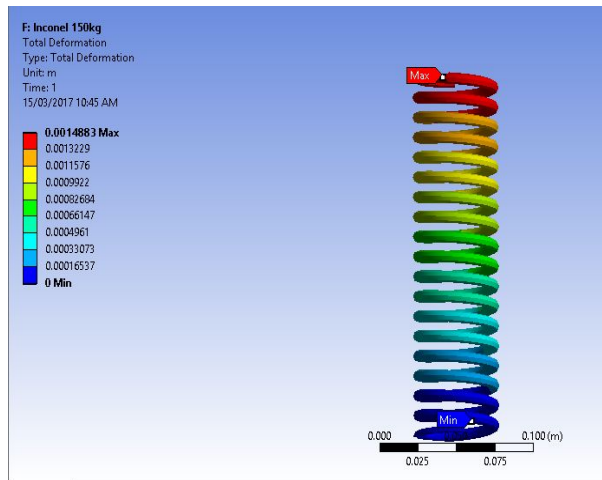


Fig.6 load (150 kg) vs deflection.

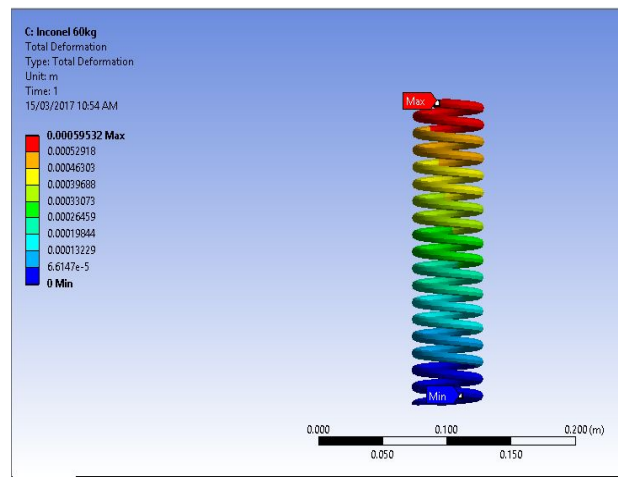


Fig.9 load (60 kg) vs deflection.

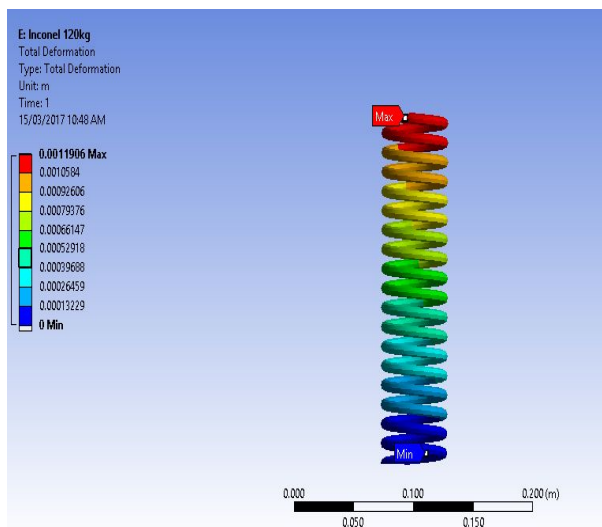


Fig.7 load (120 kg) vs deflection.

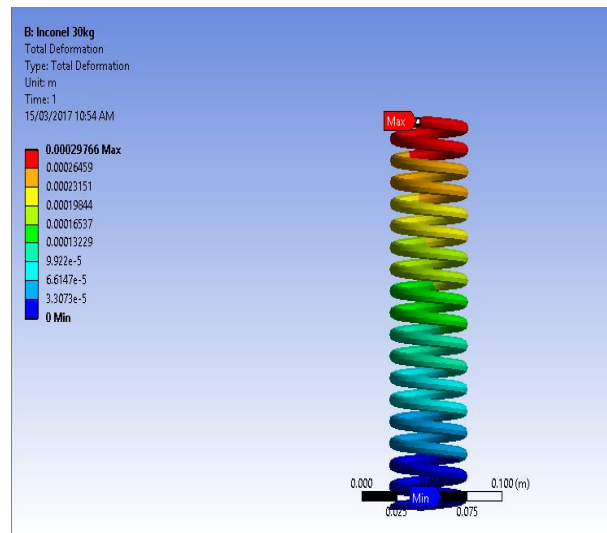


Fig.10 load (30 kg) vs deflection.

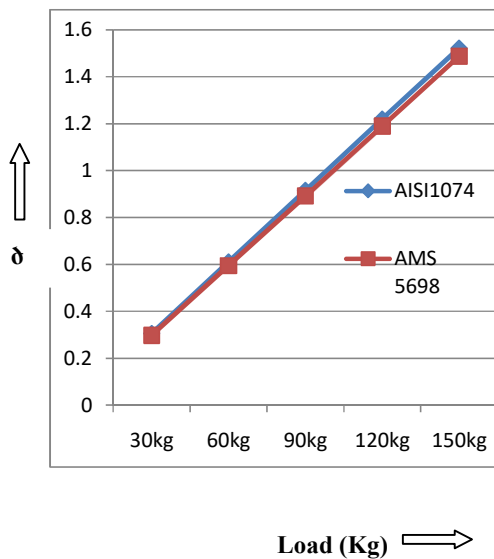
5.3 Analysis of coil spring by software ANSYS 16.2

Table No. 5.1 load vs deflection

Material Loads	Deflections at various loads, mm			
	AISI 1074	AST M A231	AST M A229	AM S 5698
30kg	0.306	0.303	0.307	0.29
60kg	0.611	0.606	0.615	0.59
90kg	0.916	0.909	0.923	0.89
120kg	1.22	1.212	1.230	1.19
150	1.523	1.515	1.53	1.48

VI. CONCLUSION

- 1) From theoretical analysis the deflection for the En 42 at 150 kg is 76.521 mm and for Inconel X 750 is 74.493mm.
- 2) From FEA analysis the deflection for the En 42 at 150 kg is 1.523 mm and for Inconel X 750 is 1.48mm.
- 3) From above both analysis we conclude that Inconel X 750 has more load carrying capacity y so that it deflects less than other coil spring and existing coil spring.
- 4) From graph we conclude that Inconel X 750 material is better than existing material (EN 42).



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